# **Opto-Mechanics and Cryostat**



### Instrument Overview



- Estimated weight: 751 lbs
- Estimated Hold time at zenith: 2.2 days

		(11.)	ī
	item	mass (lb)	
Optical Bench Assembly	Optical Bench	193	
	Image Rotator Mechanism	4.7	
	Cross Disperser Mechanism	19.4	
	Radiation Shield	8.2	
	Slit Mechanism	5.2	
	Order Sorting Mechanism	5.2	
	Immersion Grating Mechanism	5.2	
	Filter Wheel Mechanism	5.6	
	Detector, H2RG	6.6	
	Detector, Alladin	1.2	
	LN Can w/12L LN	40	
	Optics + Mounts	20	
			314.3
Cryostat Assembly	Cryostat	230	
	Trusses	8.9	
	Cryo Cooler	30	
	He Lines	20	
	Copper Flanges	5	
	Aladdin Controller	13.6	
	H2RG Controller	21.4	
			328.9
	Telescope Interface	108	108
		Total Est	751.2

## Instrument Layout – Fore Optics



# Fore Optics



#### Instrument Layout – Spectrograph



# Spectrograph Optics



# Cooling Approach



LN Dewar

- Cools the bench
- 12 L Capacity at zenith
- 2.2 Days hold time
- Cryo Cooler
  - First stage cools the radiation shield
  - Second stage cools two imagers

#### Cryostat – MeyerTool & Manufacturing

- Meyer Tool & Manufacturing experienced in large cryostat fabrication
  - South Pole Telescope Cryostat
  - Pocketed Walls
  - Aluminum Welded Construction





- NEWFIRM Dewar
- Aluminum Welded Construction
- Powder Coated

#### Cryostat – MeyerTool & Manufacturing



- AURA One Degree Instrument (ODI)
- Vacuum rated welds
- O-ring grooves with dovetail
- Large Capacity Milling Machines
- Vacuum testing capability





# Telescope Interface

- Three point interface to the Multiple Instrument Mount (MIM Cart) to facilitate final alignment to the telescope
- Gussets provide open access to the calibration optics and cryostat window





#### Trusses

- Three truss configuration (Titanium)
- Truss cross section optimized for maximum strength and thermal resistance
- Approximately 5W combined thermal load conducting through the trusses





# Flexure Study

Requirement: Co-alignment of the cold stop and telescope exit pupil to within 1% of their diameters...

*This is equivalent to 2.4mm image displacement at the secondary* 



- Worst case gravity vectors used (60° from zenith)
- The resulting angular deflection of the optical axis is expressed as image displacement at the secondary
- Total contribution is 0.45 mm at the most -20% of the allowable displacement



# **Optical Bench**

- Two sided bench
- Milled from a single billet
- Pocketed for weight reduction



#### Spectrograph Side

Fore Optics Side

## **Optic Mounts -** *Doublets*



#### Spectrograph Side

Fore Optics Side

# SpeX Lens Barrel Design

Used on SpeX with good results



## Doublet Lens Barrel Concept

Proven design approach Used on SpeX instrument

#### **Three Axial Support Pads**

- Cone geometry provides tangentline contact with convex lenses
- Spring preloaded





#### Two Radial Support Pads

- Same diameter for all lenses and spacers
- Cylindrical geometry
- Spring preloaded

### Doublet Lens Barrel Concept Lens and Spacer Stack

Axial Preload Spacer

- Protects lens from rubbing contact with spring clips
- Same pad geometry as lens spacers



Lens Spacer

- Three pad contact on front and back
- Cone geometry on pads for tangential-line contact
- Flat geometry when contacting bevels
- Small gaps allow evacuation

## Doublet Lens Barrel Concept Radial Spring Preload

#### **Radial Spring**

- Ladder Rung Design
- 6AI4V Titanium (grade 5)
- One rung per lens/spacer



#### **Externally Installed**

- Captured in the "spring well"
- Preload generated by the spring cap
- Avoids chipping lens edges



Spring Cap

- Light Tight
- Preload bosses determine the amount of spring deflection





#### Flush External Surface



## Doublet Lens Barrel Concept Interface to Optical Bench

- Mount Pads (3x)
- Located with pins (2x)
- Pin holes are chamfered to allow for mushroom effect around the pin base
- Can be shimmed if necessary
- Same approach used in SpeX



#### Optic Mounts – Slit Wheel Optics Group



#### Spectrograph Side

Fore Optics Side

## **Slit Viewer Optics**

- Group of optics above the slit wheel
- Need access to the Slit Wheel
- Need access to the Dekker Mechanism
- Need to be able to remove and install the optics group without requiring realignment



## Slit Viewer Optics - Subplate





- Four optic mounts
  - Two mirrors
  - Two doublets
  - Individually shimmable
- The optics group can be removed as an assembly
- Realignment not required
- Spherolinder Kinematic Mounts (3x)



## Slit Viewer Optics - Spherolinders

- Sphere on top, Cylinder on bottom
- 25mm sphere capable of 2 tons per spherolinder
- Hardened 440C Steel
- COTS assembly with mating cone block, V block, and preloading connector.





### **OAP** Mirrors



- Two OAP mirrors
- Mirrors and Mount to be fabricated by the optics vendor
- Mirrors will be aligned to the mount at the vendor's facility

## OAP Mount Concept

- Mirror/Mount interface uses three pads and two pins
- Mount/Bench interface uses three pads and two pins
- Wide footprint for stability and shimming resolution









## Spectrograph Camera Lens



- ISHELL has three large lenses in the Spectrograph Camera Lens
  - $\overline{\mathcal{Q}} \sim 100 \text{mm}$  each
  - V ~ 96 mm<sup>3</sup> each
  - m ~ 250 g each

Previous designs used lenses no larger than 80mm

### Spectrograph Camera Lens Barrel Mount Concept

- 3 axial and 2 radial mount pads per lens
- External coil springs hold the lens in place.
  - One per axial pad (3x)
  - One for radial loading
  - Precise holding force (10 G's or 2.5 kg)
  - Easier assembly
- Two piece lens barrel used for manufacturability
  - Precision locating features at the split
  - All lens reference surfaces are cut on a single lathe setup



### LiF Lens – Thermal Stress Analysis

NIRCam-JWST

LiF Lens is the most susceptible to fracture – selected for analysis

- FEA transient thermal analysis
- FEA internal stress analysis
- NIRCam analysis used for reference

NIRCam same sized LiF element (94 mm)





### LiF Lens – Thermal Analysis

- Symmetry used to simplify the analysis
- Contact resistance modeled at mount points (400 W/m<sup>2</sup>·K)
- Lens cell cooled at a rate of 0.5 ° K/min from 298 °K down to 73 °K.
- Includes radiative and conductive effects



- Max temp gradient ~ 3.6 ° K across the lens.
- Dominated by conductive heat transfer
- Total cooling time of 500 min (8.3 hrs).
  - Similar to SpeX

## LiF Lens – Stress Analysis

- Stress analysis based on output from thermal analysis
- NIRCam used a criteria of "maximum resolved shear stress" < 2.0 MPa.

- Largest stresses occur at 7.5 hrs
- Calculated shear stresses are ~1.25 MPa
- Mount pads need to be sized to reduce the additional contribution from contact stress.
- Risk Reduction: Plan to use a LiF blank for verification



## Schedule

