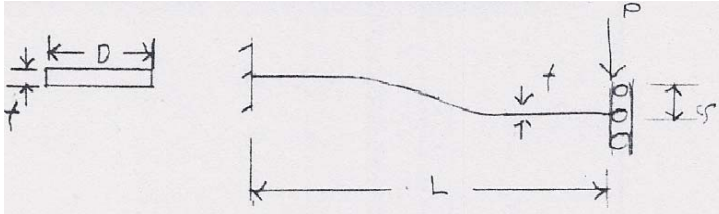


FOCUS STAGE CALCULATIONS (SPEX)

1) BENDING STRESS / DEFLECTION



$$t_b := 0.5\text{mm}$$

Beam Thickness

$$D_b := 25\text{mm}$$

Beam Depth

$$L_b := 48\text{mm}$$

Beam Length

$$F_{\text{stage}} := 32\text{N}$$

Force applied on the Focus Stage

$$N_b := 8$$

Number of Beams of the Focus Stage

$$E_b := 71700\text{N}\cdot\text{mm}^{-2}$$

Youngs Modulus: Aluminium 7075-T6

$$\sigma_y := 503\text{MPa}$$

Tensile Strength, Yield

$$P(F_{\text{stage}}) := \frac{F_{\text{stage}}}{N_b}$$

Force applied on a single Beam

$$I_b := \frac{D_b \cdot t_b^3}{12} = 2.604 \times 10^{-13} \cdot \text{m}^4$$

Beam Inertia

$$y_b := \frac{t_b}{2}$$

Distance from Neutral Axis

$$M_b := P(F_{\text{stage}}) \cdot \frac{L_b}{2} = 96 \cdot \text{N}\cdot\text{mm}$$

Bending Moment at the end of the Beam

$$\sigma_{Bb} := \frac{M_b \cdot y_b}{I_b} = 92.16 \cdot \text{MPa}$$

Bending Stress at the end of the Beam

$$\delta_b := \frac{P(F_{\text{stage}}) \cdot L_b^3}{12 \cdot E_b \cdot I_b} = 1.97431 \cdot \text{mm}$$

Deflection at the end of the Beam

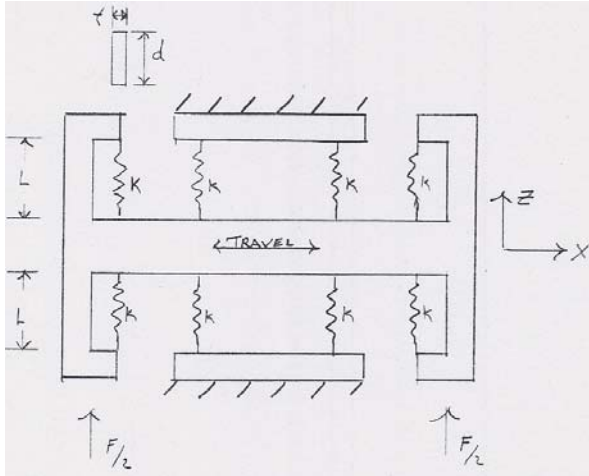
$$\delta_{\text{stage}} := 2 \cdot \delta_b = 3.949 \cdot \text{mm}$$

Travel of the Stage resulting from applied Force

$$\frac{\sigma_y}{\sigma_{Bb}} = 5.458$$

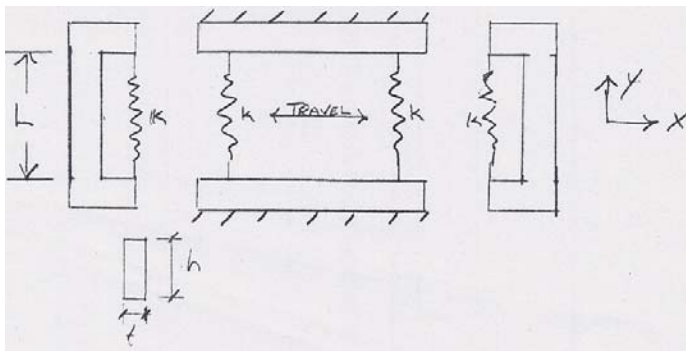
2) STIFFNESS

2.1) Perpendicular to Focus Direction (Along Z-axis):



$$K_{eq_Z} := 8 \cdot \frac{t_b \cdot D_b \cdot E_b}{L_b} = 1.494 \times 10^5 \cdot \frac{N}{mm}$$

2.2) Perpendicular to Focus Direction (Along Y-axis):



$$H_b := 38mm \quad (\text{Beam Height})$$

$$K_{eq_Y} := 8 \cdot \frac{t_b \cdot H_b \cdot E_b}{L_b} = 2.271 \times 10^5 \cdot \frac{N}{mm}$$

3) BUCKLING LOAD

$$A_b := D_b \cdot t_b$$

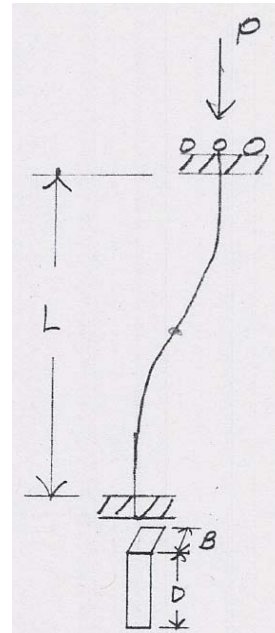
$$\rho_b := \sqrt{\left(\frac{I_b}{A_b}\right)} = 1.443 \times 10^{-4} \text{ m}$$

$$F_{BL} := \frac{(\pi^2 \cdot E_b \cdot A_b \cdot \rho_b^2)}{L_b^2} = 79.984 \text{ N}$$

Beam Section Area

Slenderness Ratio

Critical Buckling Load



4) DRIVE

Travel := 4mm

$s_p := -2\text{mm}, -1.9\text{mm} \dots 2\text{mm}$

$$F_s(s_p) := |s_p| \cdot \left(6 \cdot E_b \cdot I_b \cdot \frac{N_b}{L_b^3} \right)$$

$F_{\text{nut}} := 80\text{N}$

$F_{\text{load}} := 20\text{N}$

$$F_{\text{Is}}(s_p) := F_{\text{nut}} + (F_{\text{load}} + F_s(s_p))$$

$F_{\text{Is}}(-2\text{mm}) = 116.208\text{ N}$

Total Stage travel

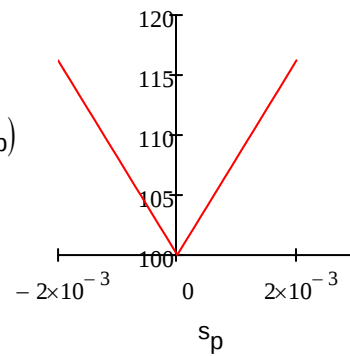
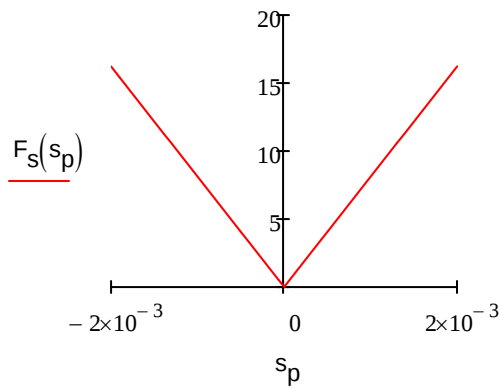
Stage position

Force to move the Flex Stage

Lead Screw Nut Preload

Load to displace

Force applied on the lead screw thread



$D_{\text{Is}} := 12.065\text{mm}$

Lead screw lead diameter

$L_{\text{Is}} := 1.27\text{mm}$

Lead screw lead

$\mu_{\text{Is}} := 0.2$

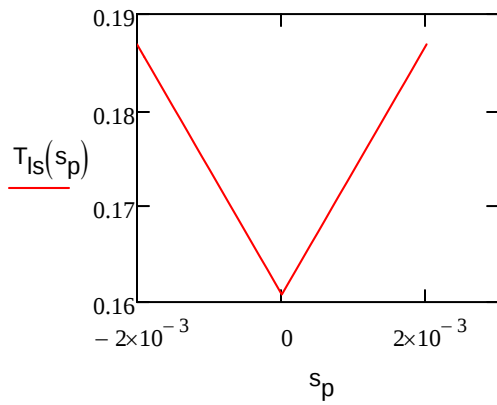
Lead screw friction coefficient (nut/screw)

$\alpha_{\text{th}} := 30\text{deg}$

ACME screw thread angle

$$T_{\text{Is}}(s_p) := \left(F_{\text{Is}}(s_p) \cdot \frac{D_{\text{Is}}}{2} \right) \cdot \left(\frac{L_{\text{Is}} + \pi \cdot \mu_{\text{Is}} \cdot \frac{D_{\text{Is}}}{\cos(\alpha_{\text{th}})}}{\pi \cdot D_{\text{Is}} - \mu_{\text{Is}} \cdot \frac{L_{\text{Is}}}{\cos(\alpha_{\text{th}})}} \right)$$

$T_{\text{Is}}(2\text{mm}) = 186.83 \cdot \text{N} \cdot \text{mm}$



$D_w := 11.4\text{mm}$ Worm Pitch Diameter

$D_{wg} := 24\text{mm}$ Worm Gear Pitch Diameter

$\gamma_w := 4.75\text{deg}$ Worm Lead Angle

$\alpha_{nw} := 14.5\text{deg}$ Pressure Angle

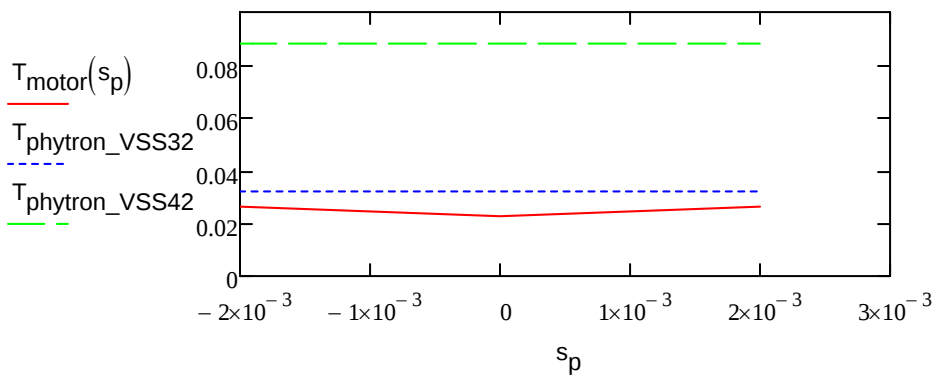
$\mu_w := 0.2$ Friction Coeff

$$T_{\text{motor}}(s_p) := T_{\text{Is}}(s_p) \cdot \left(\frac{D_w}{D_{wg}} \right) \cdot \left(\frac{\cos(\alpha_{nw}) \cdot \tan(\gamma_w + \mu_w)}{\cos(\alpha_{nw}) - \mu_w \cdot \tan(\gamma_w)} \right) \quad \text{Minimum necessary torque to drive the Focus Stage}$$

$T_{\text{motor}}(2\text{mm}) = 26.248 \cdot \text{N} \cdot \text{mm}$ Maximum Torque

$T_{\text{phytron_VSS32}} := 32\text{N} \cdot \text{mm}$ Phytron VSS32 Max. Torque

$T_{\text{phytron_VSS42}} := 88\text{N} \cdot \text{mm}$ Phytron VSS42 Max. Torque



$$R_{ls} := \frac{\text{Travel}}{L_{ls}}$$

Number of Lead screw revolutions for full travel

$$\text{Ratio}_w := 30$$

Reduction Ratio of Worm

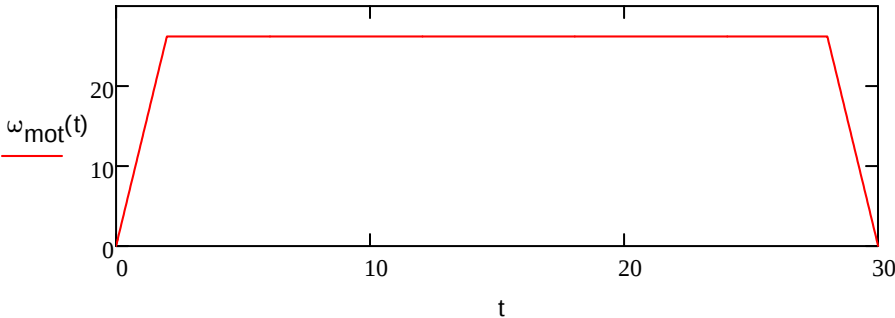
$$\text{Travel}_{rot} := \text{Ratio}_w \cdot R_{ls} = 94.488$$

Number of Motor Revolutions for full travel

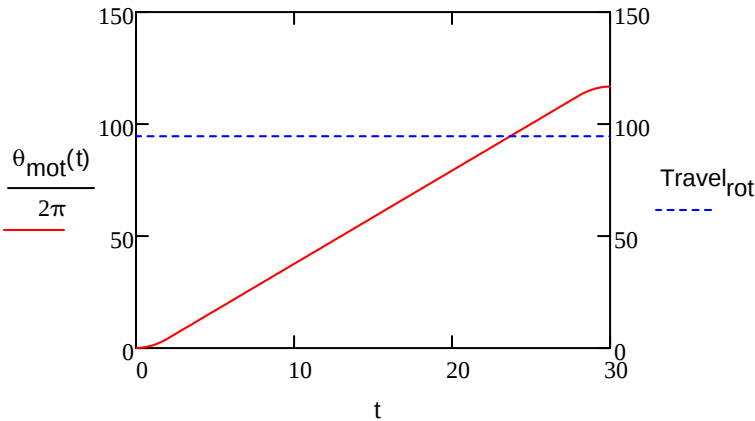
$$t_{t1} := 2s \qquad t_{t2} := 28s \qquad t_{tt} := 30s$$

$$\omega_{mot_max} := 250rpm$$

$$\omega_{mot}(t) := \text{if} \left[t \leq 0, 0, \text{if} \left[t < t_{t1}, \omega_{mot_max} \cdot \frac{t}{t_{t1}}, \text{if} \left[t < t_{t2}, \omega_{mot_max}, \text{if} \left[t < t_{tt}, \omega_{mot_max} \cdot \frac{(t_{tt} - t)}{(t_{tt} - t_{t2})}, 0 \right] \right] \right] \right]$$



$$\theta_{mot}(t) := \int_0^t \omega_{mot}(t) \, dt$$



$$W_{bl} := 0.006 \text{ in}$$

Typical Backlash for a Commercial Grade Worm (0.003" - 0.006")

$$\alpha_{bl} := \text{atan}\left(\frac{2W_{bl}}{D_{wg}}\right) = 0.728 \cdot \text{deg}$$

Angular Backlash

$$\epsilon_{ls} := L_{ls} \cdot \frac{\alpha_{bl}}{2\pi} = 2.567 \cdot \mu\text{m}$$

Resulting position error of Lead Screw